

METHOD FOR CONTROLLING A FUEL METERING SYSTEM  
OF AN INTERNAL COMBUSTION ENGINEFIELD OF THE INVENTION

The present invention relates to a method for controlling a fuel metering system of an internal combustion engine.

5 BACKGROUND INFORMATION

A method and device for controlling a fuel metering system of an internal combustion engine is discussed, for example, in German patent document no. 199 45 618. In this method, the activation duration of at least one electrically operated valve determines the fuel quantity to be injected. The minimum activation duration during which fuel is only just injected is determined in certain operating states. The activation duration is increased or reduced, starting at an initial value. The activation duration during which a signal undergoes a change is stored as the minimum activation duration. The signal used is a quantity characterizing the uniformity of rotation, an output signal of a lambda sensor, or an output signal of an ion current sensor. This method allows for an injection quantity drift over the lifetime of the fuel injector in the range of the pilot injection.

Non-prepublished German patent document no. 102 15 610 discusses a system and method for correcting the injection behavior of injectors, in which, in order to increase the product output, an injector fuel-quantity compensation is carried out at several test points, which may be 4 test points, i.e., during the pilot injection, during idle operation, at the emission point, and during full-load operation.

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This injector fuel-quantity compensation is necessary because injectors of that kind have different fuel quantity maps due to their mechanical manufacturing tolerances. A "fuel quantity map" is understood to be the relationship between the  
5 injection quantity, the rail pressure, and the activation time (activation duration). As a consequence, in spite of the electrically defined control, each individual injector fills the combustion chamber with different fuel quantities.

10 In fact, to achieve as low a fuel consumption as possible while complying with strict exhaust emission standards, the injectors may only have very small tolerances in terms of the injection quantity during operation. These small tolerances required cannot be met due to the mechanical manufacturing  
15 tolerances. In order to nevertheless guarantee a defined injection quantity of the injectors, the injectors are measured for their injection quantity at characteristic operating points or test points after manufacture, and are classified. The respective class must be known to the engine  
20 control unit during the operation of the internal combustion engine so that the control can be adapted to the particular characteristics of the class specifically for each injector. The class information is stored on the injector, for example, by different codes such as a bar code, by resistors on the  
25 injector, or by plaintext on the injector.

Moreover, electronic storage means, in which are stored, for example, the class information, may be provided in the injectors. These values may be read out from the injector by  
30 the control unit via an interface and used in the subsequent operation.

Over their lifetime, such common rail injectors are observed to have a fuel quantity drift, which is different for each  
35 individual injector and depends, for example, on the load

profile or on the type of injector. This fuel quantity drift has a negative effect in terms of low fuel consumption, on compliance with strict exhaust emission standards, and with respect to, for example, the noise level of the internal combustion engine. Until now, correction of the injection quantity drift over the lifetime of the injectors was only done in the pilot injection plateau using a method discussed in German patent document no. 199 45 618. In contrast, fuel quantity drifts at other operating points can only be compensated for to a very limited degree, if at all.

#### SUMMARY OF THE INVENTION

It is therefore an object of the exemplary embodiment and/or exemplary method of the present invention to provide a method which allows for fuel quantity correction, in particular, for correction of the fuel quantity drift over the lifetime of a piezoelectric common rail injector at operating points outside of the range of the pilot injection.

20 In an exemplary method for controlling injectors of a fuel metering system of an internal combustion engine of the type mentioned at the outset, this objective is achieved by the features of the method described herein.

25 The exemplary embodiment and/or exemplary method of the present invention includes determining correction values for the fuel quantity map of the injector from the difference between the activation duration during which a signal undergoes a change and the stored minimum activation duration using transfer functions which define the relationship between the minimum activation duration and the activation duration at different test points of the injector, respectively, and/or the relationship between the activation durations at several test points of the injector. Thus, in a way, the entire fuel quantity map of the injector is inferred based on the range of

the pilot injection, in which fuel quantity drifts can be determined and corrected, using transfer functions.

5 These transfer functions, in turn, may be determined during the injector fuel-quantity compensation at the test points.

10 One exemplary embodiment includes storing the transfer functions on the injector, i.e., to encode the injector with these transfer functions as well.

15 In accordance with another exemplary embodiment, the correction values are stored in an engine control unit.

#### BRIEF DESCRIPTION OF THE DRAWINGS

20 Figure 1 shows a schematic representation of a portion of a common rail system known from the prior art, in which the exemplary method according to the present invention is used.

25 Figure 2 schematically shows the injection quantity over the activation duration for different injection pressures to illustrate the exemplary method according to the present invention.

#### DETAILED DESCRIPTION

30 Figure 1 shows the high-pressure stage of a common rail accumulator injection system, of which only the main components and components that are essential for understanding the present invention will be explained in more detail hereinafter.

35 The system includes a high-pressure pump 10 which is in communication with high-pressure accumulator ("rail") 14 via a high-pressure delivery line 12. High-pressure accumulator 14 is connected to injectors 18 via further high-pressure delivery lines. In this representation, a high-pressure

delivery line 16 and an injector 18 are shown. Injector 18 is mounted in an internal combustion engine of a motor vehicle. The system shown is controlled by an engine control unit 20, which controls, in particular, injector 18.

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Injector 18 is provided with a device 22 for storing information that relates to the individual injector 18. The information stored in device 22 may be taken into account by engine control unit 20 so that each injector 18 may be controlled individually. The information may consist of correction values for the fuel quantity map of injector 18. Device 22 for storing the information may be implemented, for example, as a data memory, or also as one or more electrical resistors, as a bar code, by alphanumeric coding, or the like, or also by a semiconductor integrated circuit located on injector 18. Engine control unit 20 may also have a semiconductor integrated circuit for evaluating the information stored in device 22.

20 The injection quantity metered in by each injector 18 is defined as a function of the rail pressure in a characteristic map stored in engine control unit 20, the characteristic map being determined based on several test points (pilot injection, idling, emission point, full load), which correspond to different operating states of the internal combustion engine. At each of these test points, fuel quantity compensation is carried out in a manner which is known per se and DISCUSSED in German patent document no. 102 15 610. The injection quantity is determined by the injection duration of injector 18; i.e., the time that passes between the start of injection and the end of injection.

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35 In order to enable fuel quantity metering over the entire operating range of the internal combustion engine and injector 18, the compensation values are interpolated between the nodes

defined by the test points.

Over the lifetime of injectors 18, a fuel quantity drift can be observed to the effect that the injection quantity, which 5 was originally determined by defining the start of injection and the injection duration, changes over the life of injectors 18.

To compensate for this fuel quantity drift in a common rail 10 injector, for example, in a common rail injector having a fully ballistic characteristic map without pilot injection plateau (VE), which is schematically shown in Figure 2, an injector fuel-quantity compensation (IMA) is carried out, for 15 example, at the five aforementioned test points LL (idling), VE1 (pilot injection 1), EM (emission point), VE2 (pilot injection 2), and VL (full load). In addition, transfer functions  $\ddot{U}_1 = f(VE1, LL)$ ,  $\ddot{U}_2 = f(VE2, EM)$  and  $\ddot{U}_3 = f(EM, VL)$  are determined.

20 In order to implement a fuel quantity correction over the lifetime of injector 18, then the minimum activation duration during which fuel is only just injected is determined at two operating points VE1 and VE2; the activation duration being increased or reduced starting at an initial value, and the 25 activation duration during which a signal undergoes a change being stored as the minimum activation duration  $VE1'$  and  $VE2'$ , respectively. Then, the differences  $\Delta_{11}(VE1, VE1')$  and  $\Delta_{12}(VE2, VE2')$  are determined, respectively. From these differences, correction values for the fuel quantity map of 30 the injector are now determined and stored using the transfer functions  $\ddot{U}_1$ ,  $\ddot{U}_2$ ,  $\ddot{U}_3$ , which define the relationship between minimum injection duration  $VE1$  and the injection duration during idle operation LL, the relationship between the second minimum injection duration and the injection duration at 35 emission point EM, and the relationship between different

operating points, for example, the injection duration injection duration at emission point EM and the injection duration during full-load operation VL:

5     $LL' = f (LL, \Delta t_1, \bar{U}_1)$ ,  $EM' = f (EM, \Delta t_2, \bar{U}_2)$  und  $VL' = f (VL, EM', \bar{U}_3)$ .

10   Thus, in a way, the injection durations at the further test points LL', EM', VL' of the fuel quantity map of injector 18 are inferred from the so-called "zero fuel quantity correction", i.e., from the correction of the minimum injection duration or minimum injection durations VE1, VE2 using transfer functions  $\bar{U}_1$ ,  $\bar{U}_2$ ,  $\bar{U}_3$ .

15   These transfer functions may be determined during the injector fuel-quantity compensation, or also independently of the injector fuel-quantity compensation.

20   Transfer functions  $\bar{U}_1$ ,  $\bar{U}_2$ ,  $\bar{U}_3$  may either be stored on injector 18, i.e., injector 18 may be encoded with the transfer functions, or be stored in engine control unit 20 as a characteristic curve. Advantageously, map points  $LL' = f (LL, \Delta t_1, \bar{U}_1)$ ,  $EM' = f (EM, \Delta t_2, \bar{U}_2)$  and  $VL' = f (VL, EM', \bar{U}_3)$  are corrected each time a zero fuel quantity correction is 25 carried out. In this manner, a closed fuel quantity control loop is achieved.